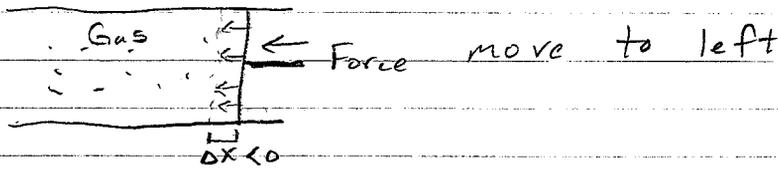


## 1.5 Compression Work

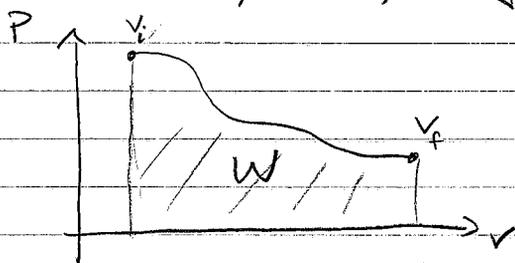


Remember work done on object  $\vec{F} \cdot d\vec{r}$

If there is no friction and you push at constant speed  
 $|\vec{F}| = P \cdot A$

$$W = \vec{F} \cdot d\vec{r} = -PA \Delta x = -P \Delta V$$

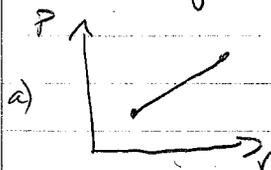
If  $\Delta V$  is negative, positive work is done on gas  
 If " positive, negative " " " " "



$$W = - \int_{V_i}^{V_f} P(V) dV$$

why does P depend on V?

Prob 1.31 1 liter of He at 1 atm goes to 3 liters so  $P(V) \propto V$



$$P(V) = \frac{1.01 \times 10^5 \text{ Pa}}{10^{-3} \text{ m}^3} V$$

b)

$$W = - \int_{V_i}^{V_f} P(V) dV = -1.01 \times 10^8 \frac{\text{Pa}}{\text{m}^3} \left( \frac{V_f^2 - V_i^2}{2} \right)$$

$$= -4.04 \times 10^2 \text{ J} = -404 \text{ J}$$

c)

$$\Delta U = \frac{3}{2} N k_B (T_f - T_i) = \frac{3}{2} (P_f V_f - P_i V_i) = 12.12 \times 10^2 \text{ J} = 1212 \text{ J}$$

d)

$$Q = \Delta U - W = 1616 \text{ J}$$

e) Put a flame under canister . . .

The compression of an ideal gas provides examples with closed expressions.

Isothermal compression -  $T = \text{constant} \Rightarrow P \propto 1/V$

$$W = - \int_{V_i}^{V_f} P dV = - \int_{V_i}^{V_f} \frac{Nk_B T}{V} dV = \boxed{Nk_B T \ln\left(\frac{V_i}{V_f}\right)}$$

Note  $\Delta U = \Delta\left(\frac{1}{2} Nk_B T\right) = 0! \Rightarrow Q = \Delta U - W = Nk_B T \ln\left(\frac{V_f}{V_i}\right)$

For ideal gas, isothermal compression gives  $Q = -W$ . That is heat input is minus the work done on gas.

Adiabatic compression  $Q = 0$  (no heat in/out)  
 $\Rightarrow \Delta U = W$

Use  $u = \frac{f}{2} Nk_B T \Rightarrow du = \frac{f}{2} Nk_B dT = -P dV$

For ideal gas  $P = \frac{N}{V} k_B T \Rightarrow \frac{f}{2} Nk_B dT = -\frac{N}{V} k_B T dV$

$$\Rightarrow \frac{f}{2} \frac{dT}{T} = -\frac{dV}{V} \Rightarrow \frac{f}{2} \ln\left(\frac{T_f}{T_i}\right) = -\ln\left(\frac{V_f}{V_i}\right) = \ln\left(\frac{V_i}{V_f}\right)$$

$$\Rightarrow \left(\frac{T_f}{T_i}\right)^{f/2} = \frac{V_i}{V_f} \Rightarrow V_f T_f^{f/2} = V_i T_i^{f/2}$$

$$VT^{f/2} = \text{constant}$$

Prob 1.35

use  $PV = Nk_B T \Rightarrow V^{(1+f/2)} P^{f/2} = \text{constant}$

raise both sides to  $\frac{2}{f}$  power  $V^{\frac{2+f}{f}} P = \text{constant}$   
 $V^\gamma P = \text{constant}$   
 $\gamma = \frac{2+f}{f}$

Prob 1.37 In diesel engine, air compressed to  $1/20$  of volume quickly.

$$\Rightarrow Q = 0 \quad T_f = T_i \left(\frac{V_i}{V_f}\right)^{\gamma} = 20^{2/5} T_i$$

$$f = 5 \text{ for } N_2 \quad T_i = 300 \text{ K} \quad T_f = 1000 \text{ K}$$

Prob 1.40 Show  $\frac{dT}{dP} = \frac{2}{f+2} \frac{T}{P}$  when gas expands adiabatically

$$VT^{f/2} = \text{constant} \quad \text{use} \quad V = \frac{Nk_B T}{P} \Rightarrow T^{(f/2+1)} P^{-1} = \text{const.}$$

$$\Rightarrow T = \text{const} P^{\frac{1}{f/2+1}} = \text{const} P^{\frac{2}{f+2}}$$

$$\frac{dT}{dP} = \frac{2}{f+2} \text{const} P^{\frac{2}{f+2}-1} = \frac{2}{f+2} \frac{T}{P}$$

From Prob 1.16  $\frac{dP}{dz} = -\frac{mg}{k_B} \frac{P}{T}$

$$\Rightarrow \frac{dT}{dz} = \frac{dP}{dz} \frac{dT}{dP} = -\frac{mg}{k_B} \frac{P}{T} \frac{2}{f+2} \frac{T}{P} = -\frac{mg}{k_B} \frac{2}{f+2}$$

use  $f=5$   $m = 4.65 \times 10^{-26} \text{ kg} \Rightarrow \frac{dT}{dz} \approx -9.4 \times 10^{-3} \frac{\text{K}}{\text{m}}$   
 $\approx -9.4 \frac{\text{K}}{\text{km}}$   
 $= -9.4 \text{ } ^\circ\text{C}/\text{km}$

If  $\frac{dT}{dz}$  decreases faster than this, then warm, low density air will rise and cold high density air will sink giving convection